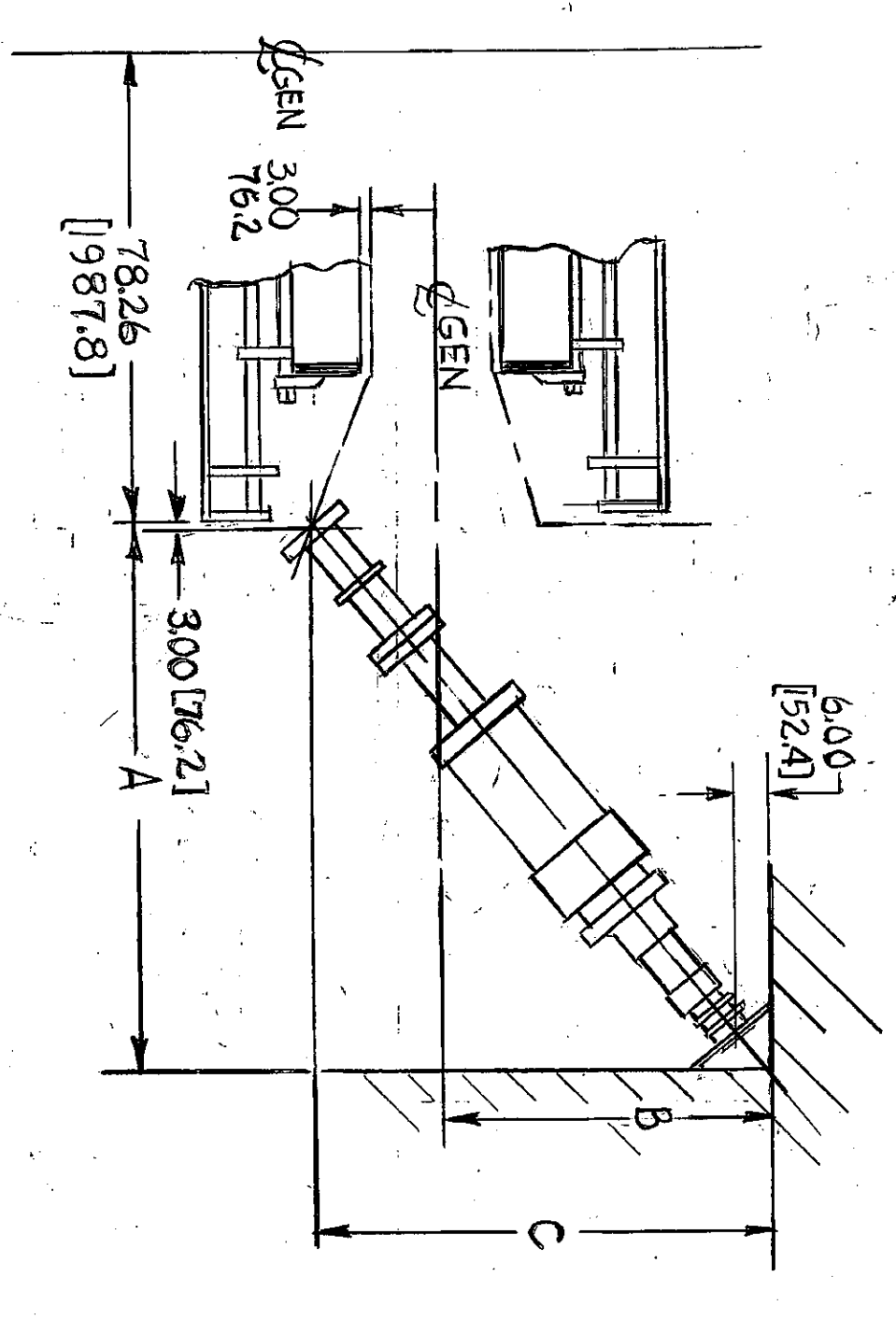
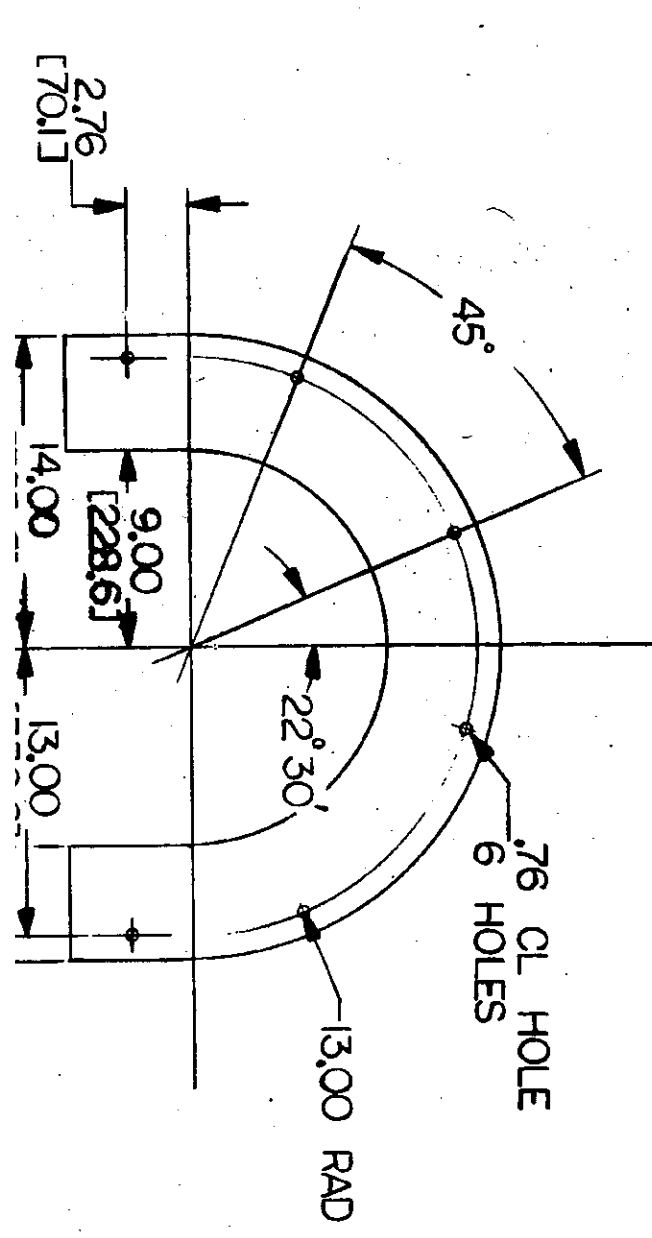
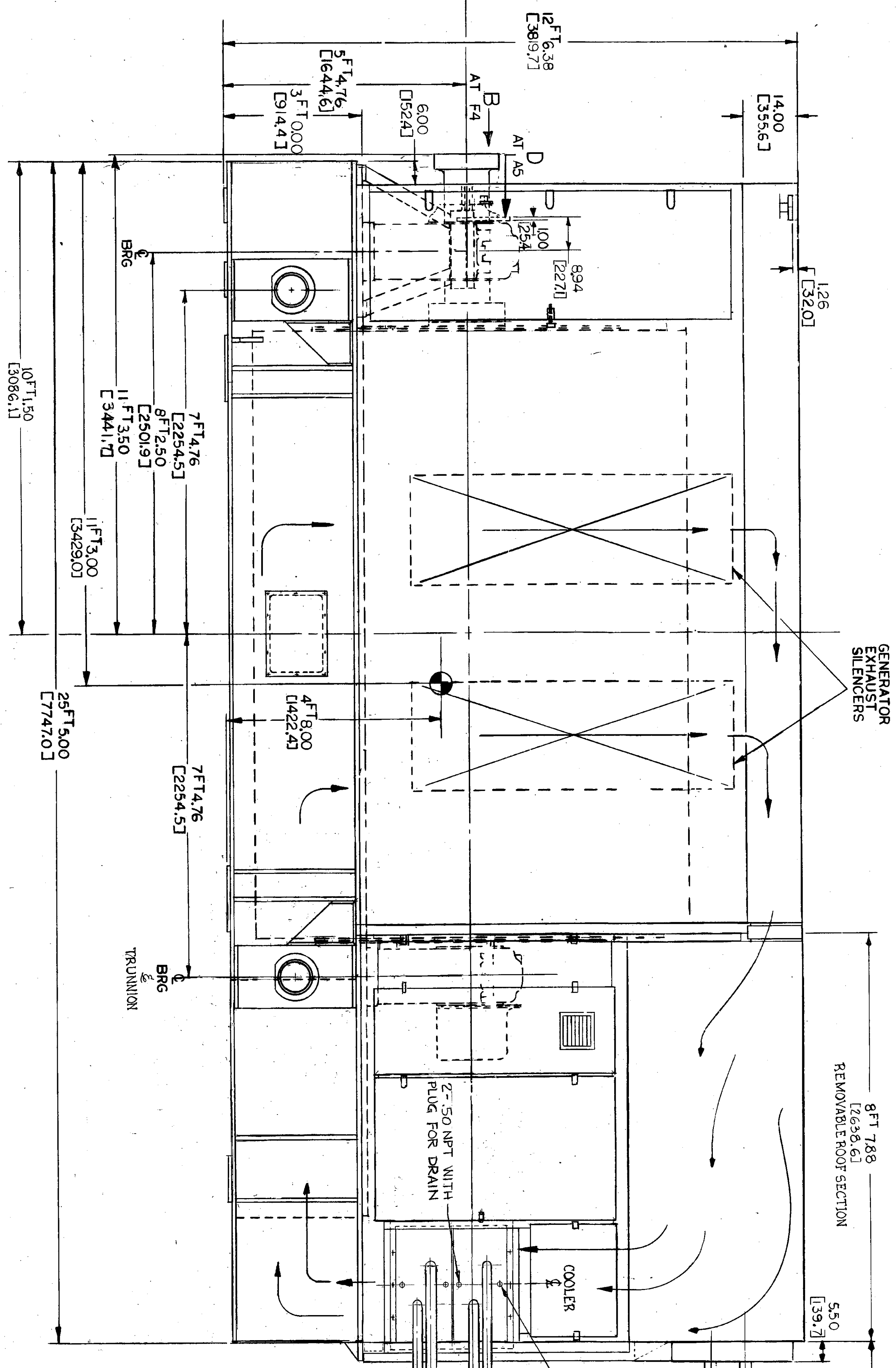
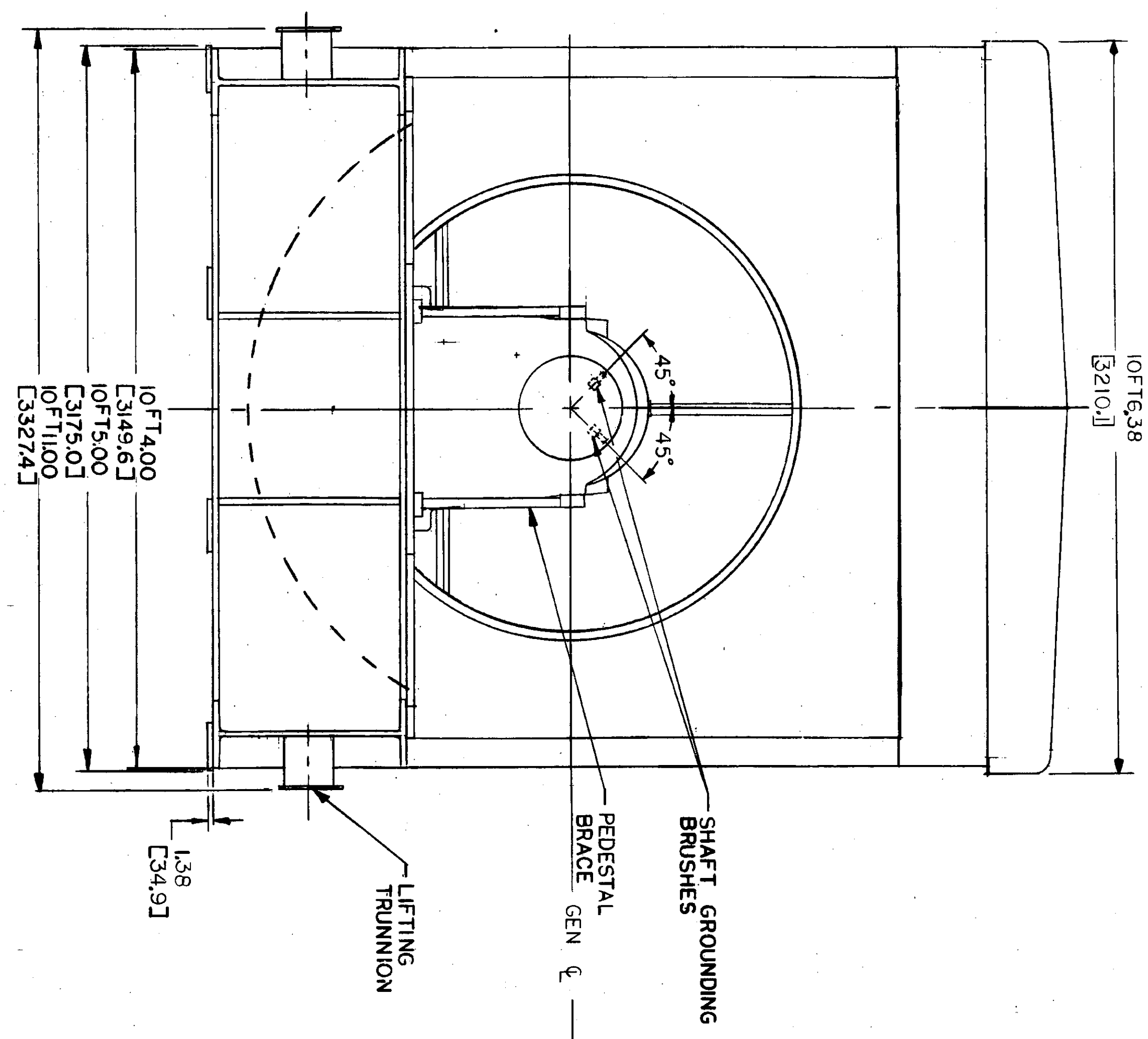
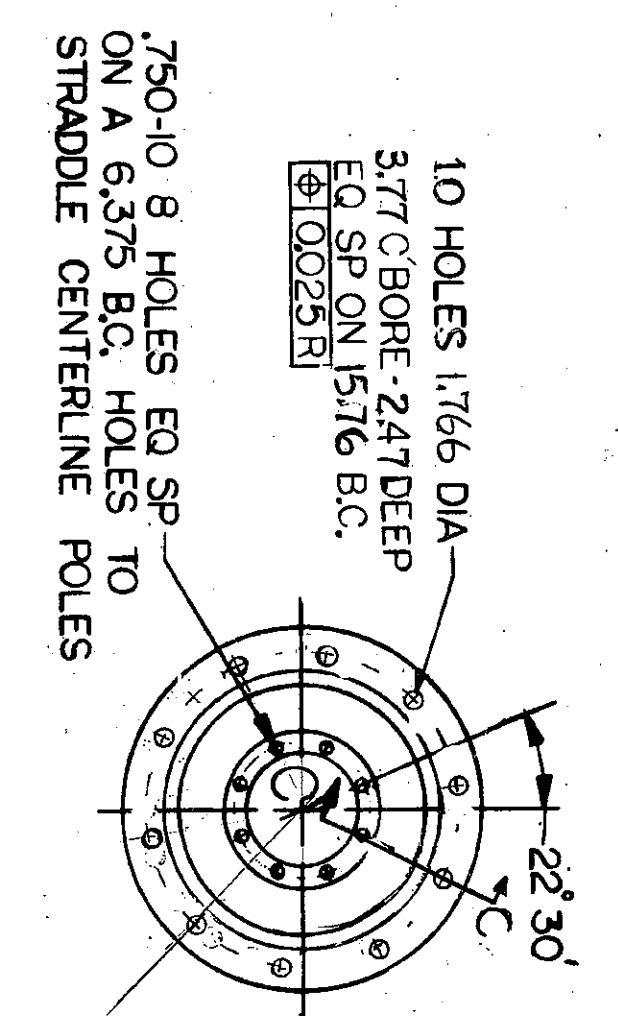
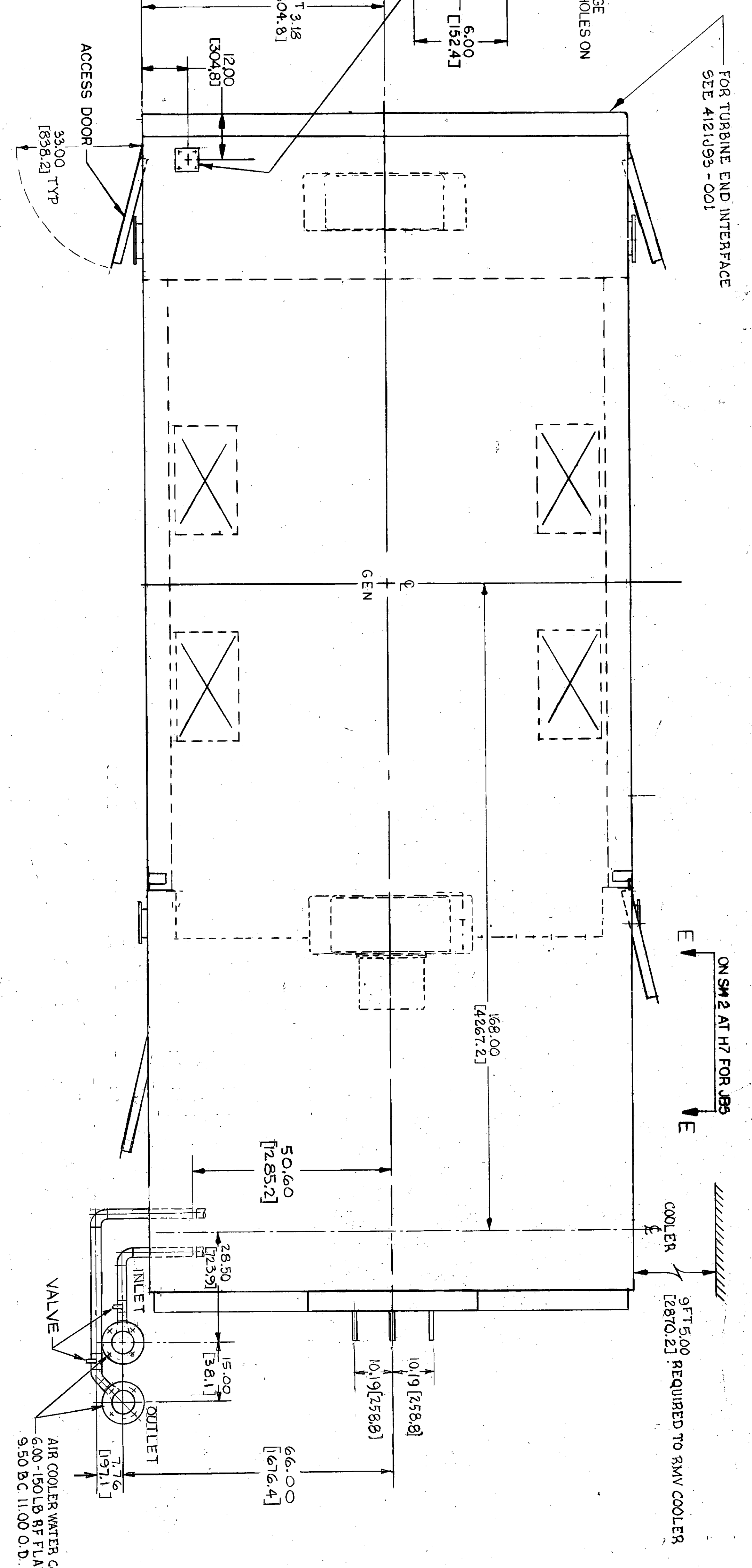
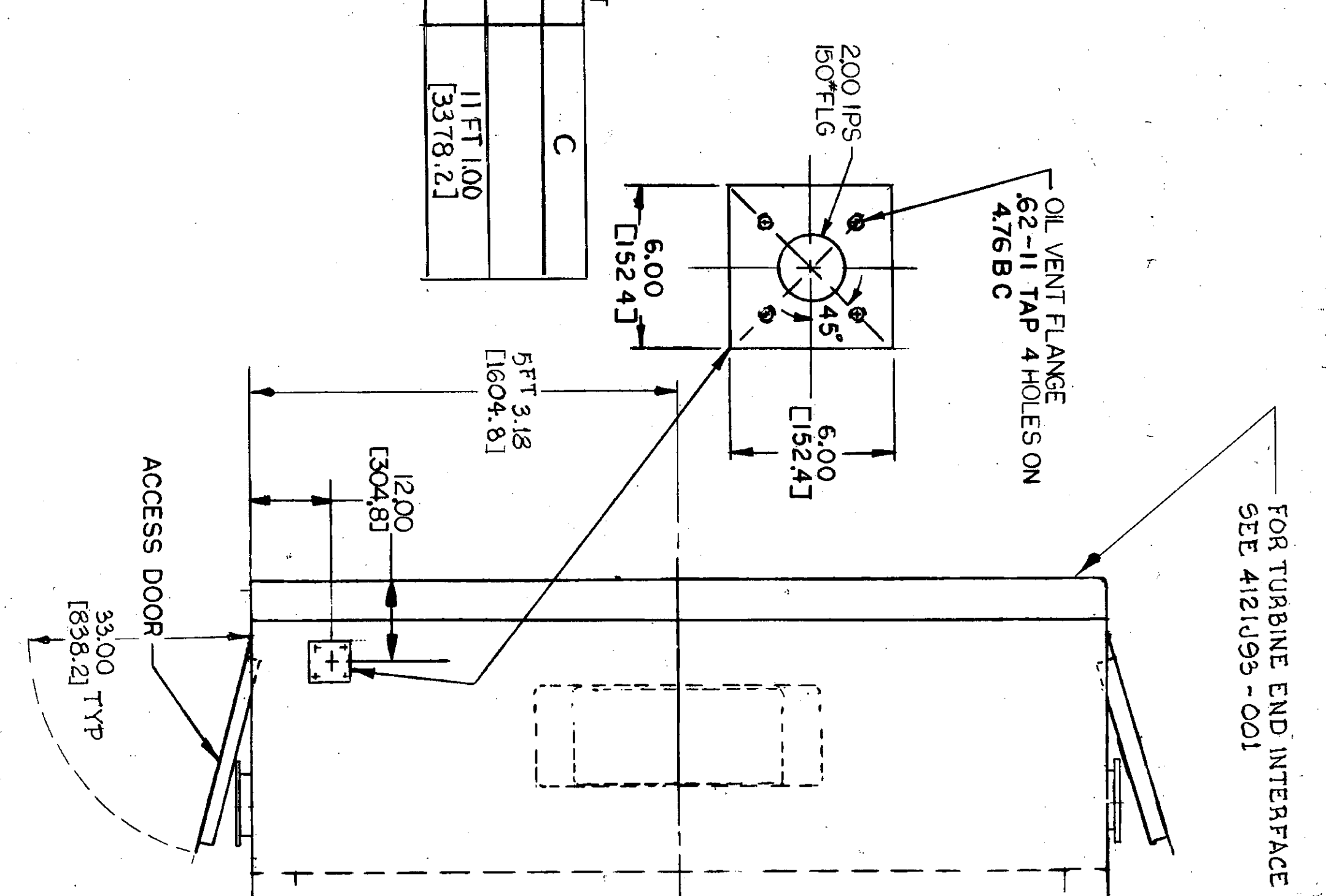


A B C D E F G H J



DISTANCE REQUIRED TO REMOVE ROTOR WITH STUB SHAFT

METHOD OF REMOVAL	A	B	C
STRAIGHT PULL	20 FT 10.00 [6350.00]	8 FT 4.00 [2540.00]	11 FT 1.00 [3378.21]
CANTING HOR'LY	17 FT 9.00 [5410.21]	8 FT 4.00 [2540.00]	11 FT 1.00 [3378.21]



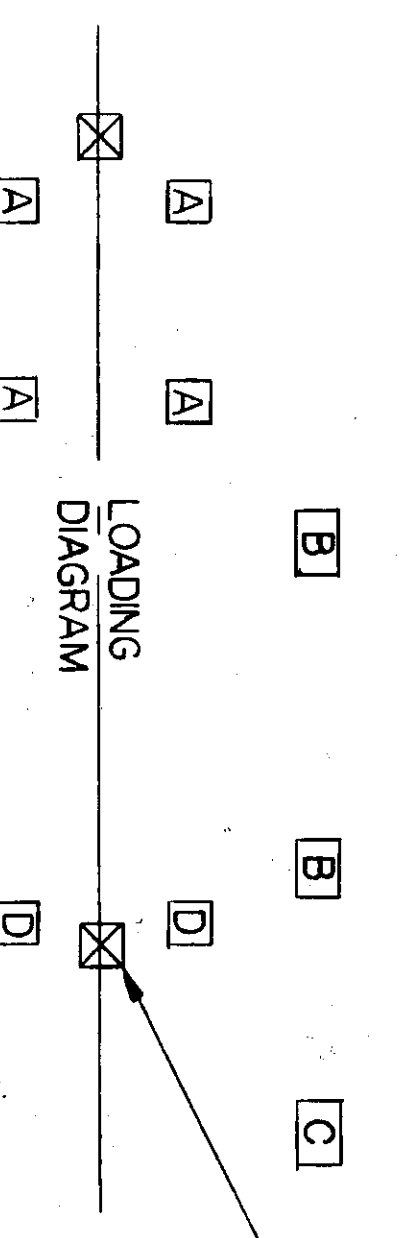
Sheet 16

12 11 10 9 8 7 6 5 4



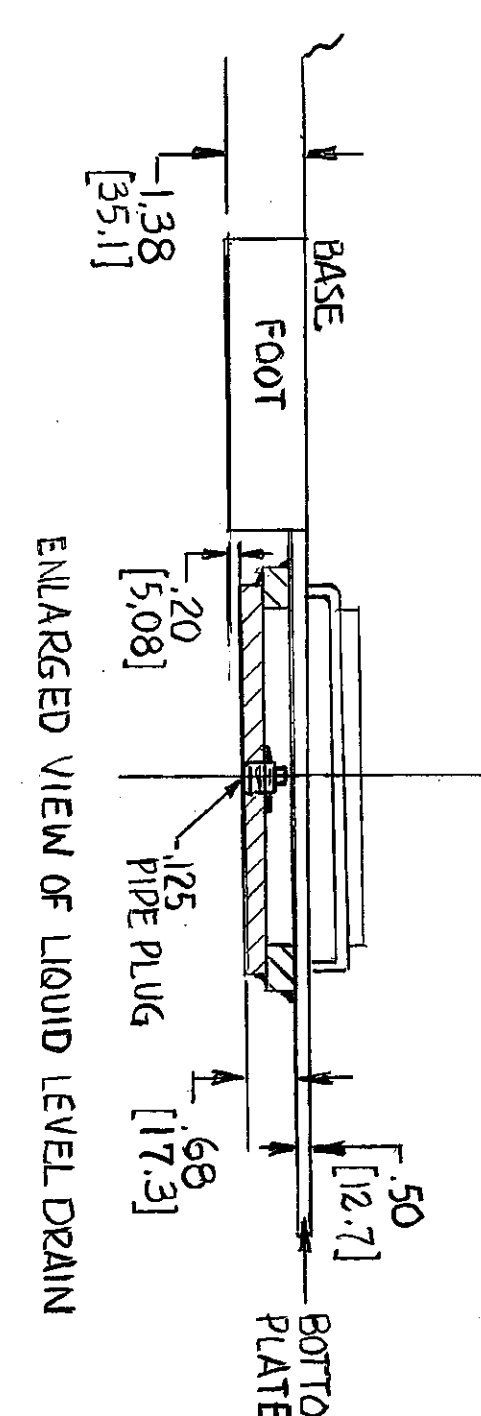
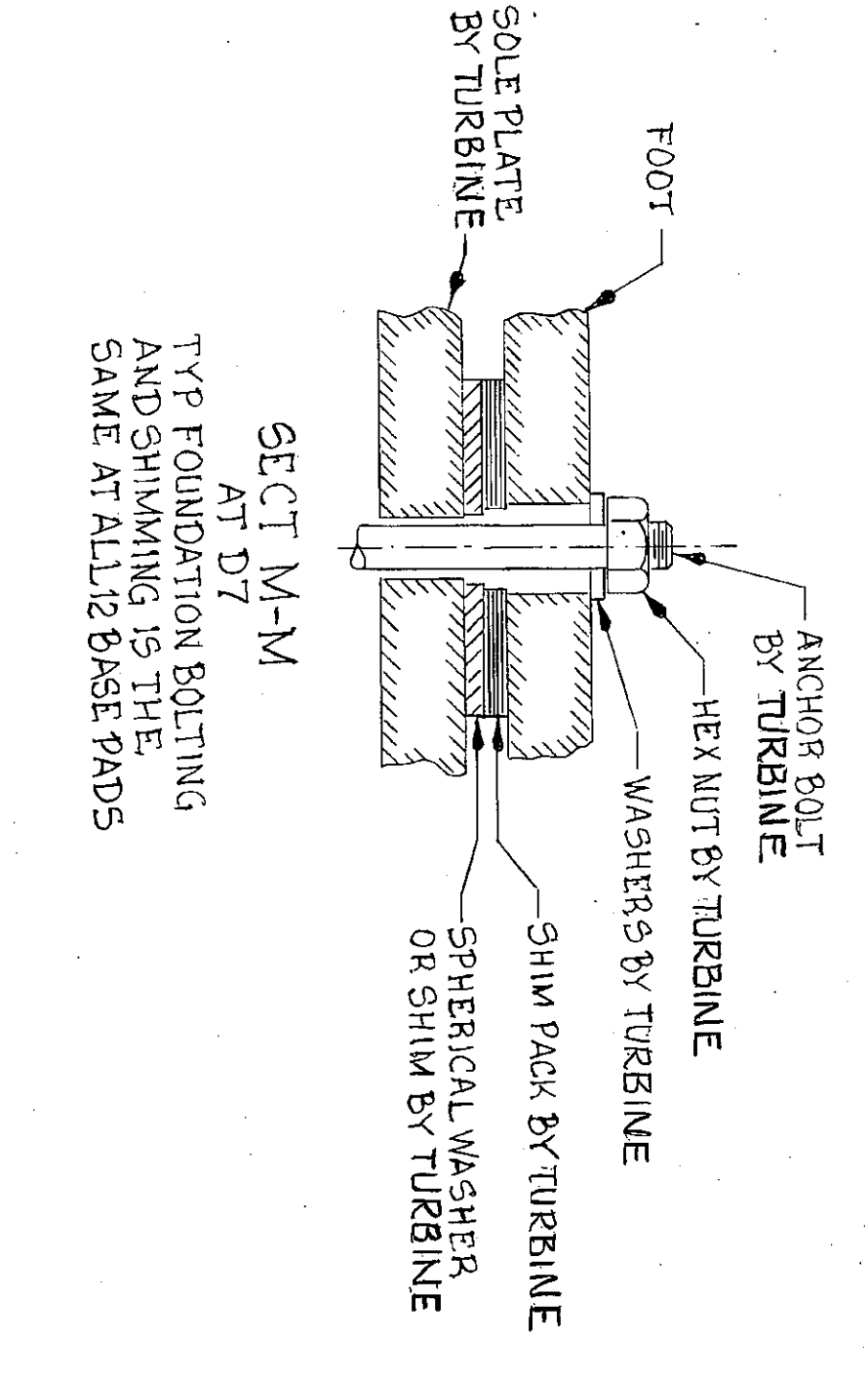
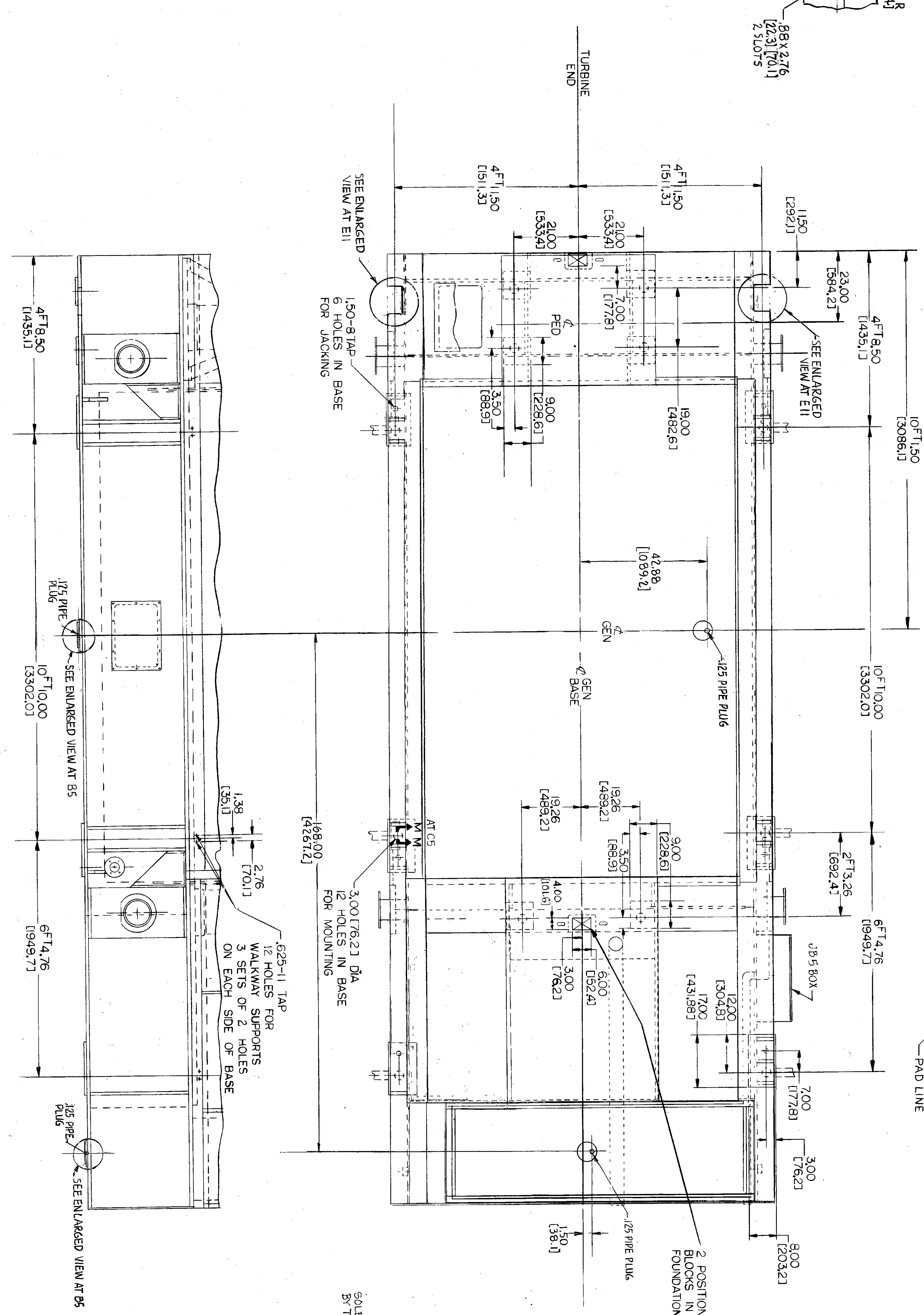
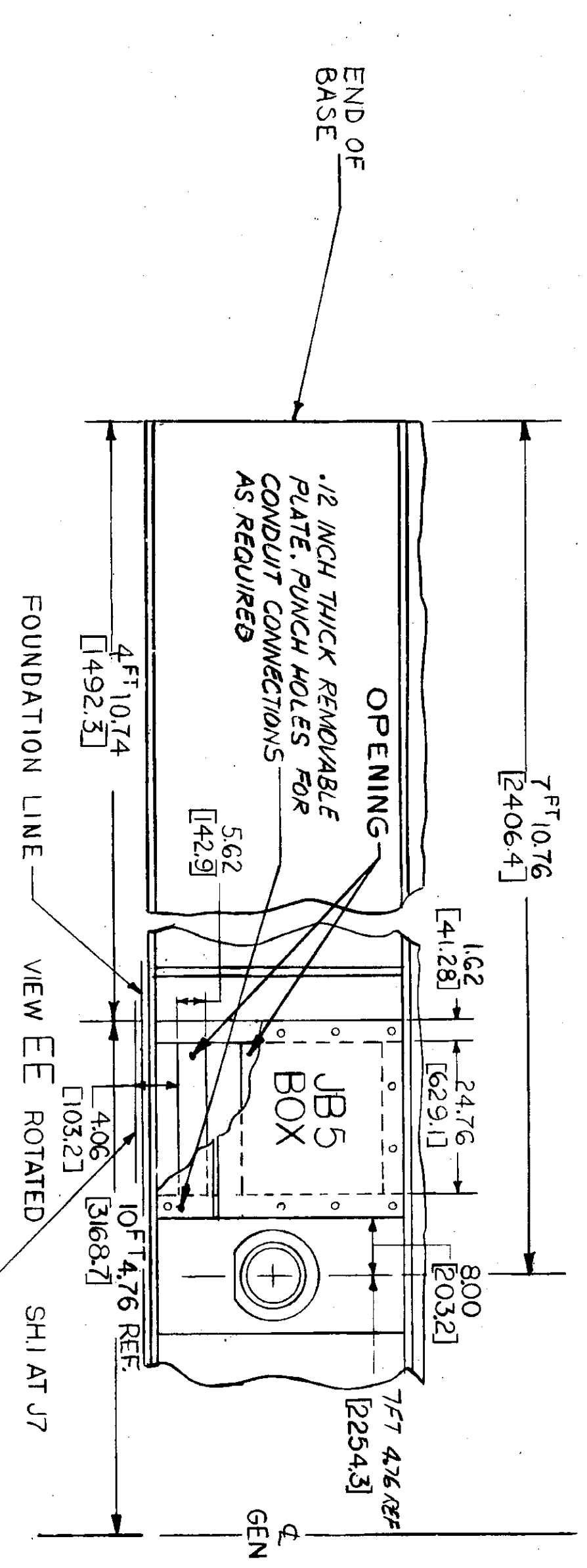
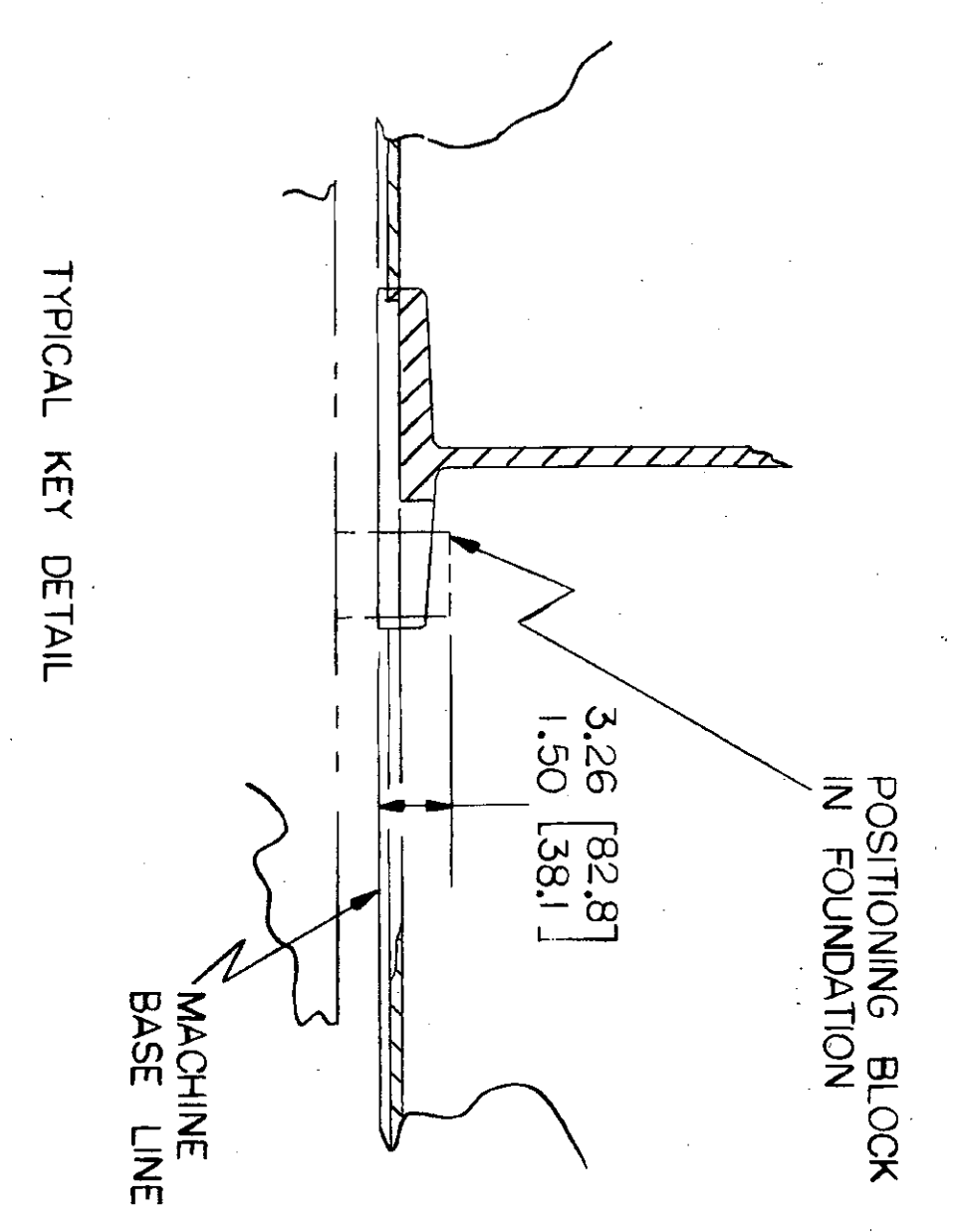
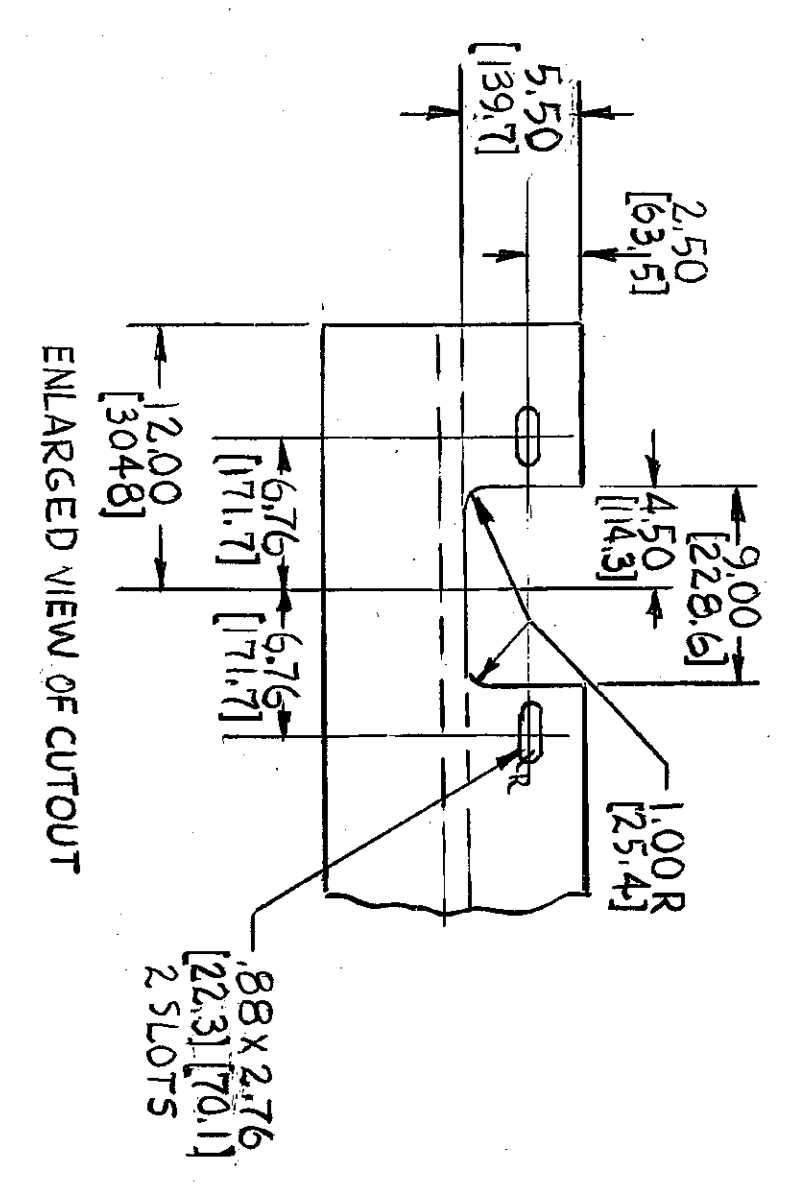






POSITIONING BLOCKS ARE TO BE DESIGNED TO WITHSTAND 24300 LBS (1113 KGS) LATERAL THRUST WITH MAXIMUM DEFLECTION AT TRANSVERSE KEY OF .002 (0.095) EACH KEY EACH WAY

APPROXIMATE FOUNDATION LOADS			
LOAD DESCRIPTION	A	B	C
STATIC POUNDS	5210	26960	5840
STATIC KILOGRAMS	2363	12229	2649
LOADING FROM SHORT CIRCUIT TORQUE	3600 POUNDS (TON EACH) ± 633 KILOGRAMS (MTS PAD)		
AXIAL LOAD	30% OF STATIC LOAD		
LATERAL LOAD	12% OF STATIC LOAD		
SEISMIC VERT.	108% OF STATIC LOAD		

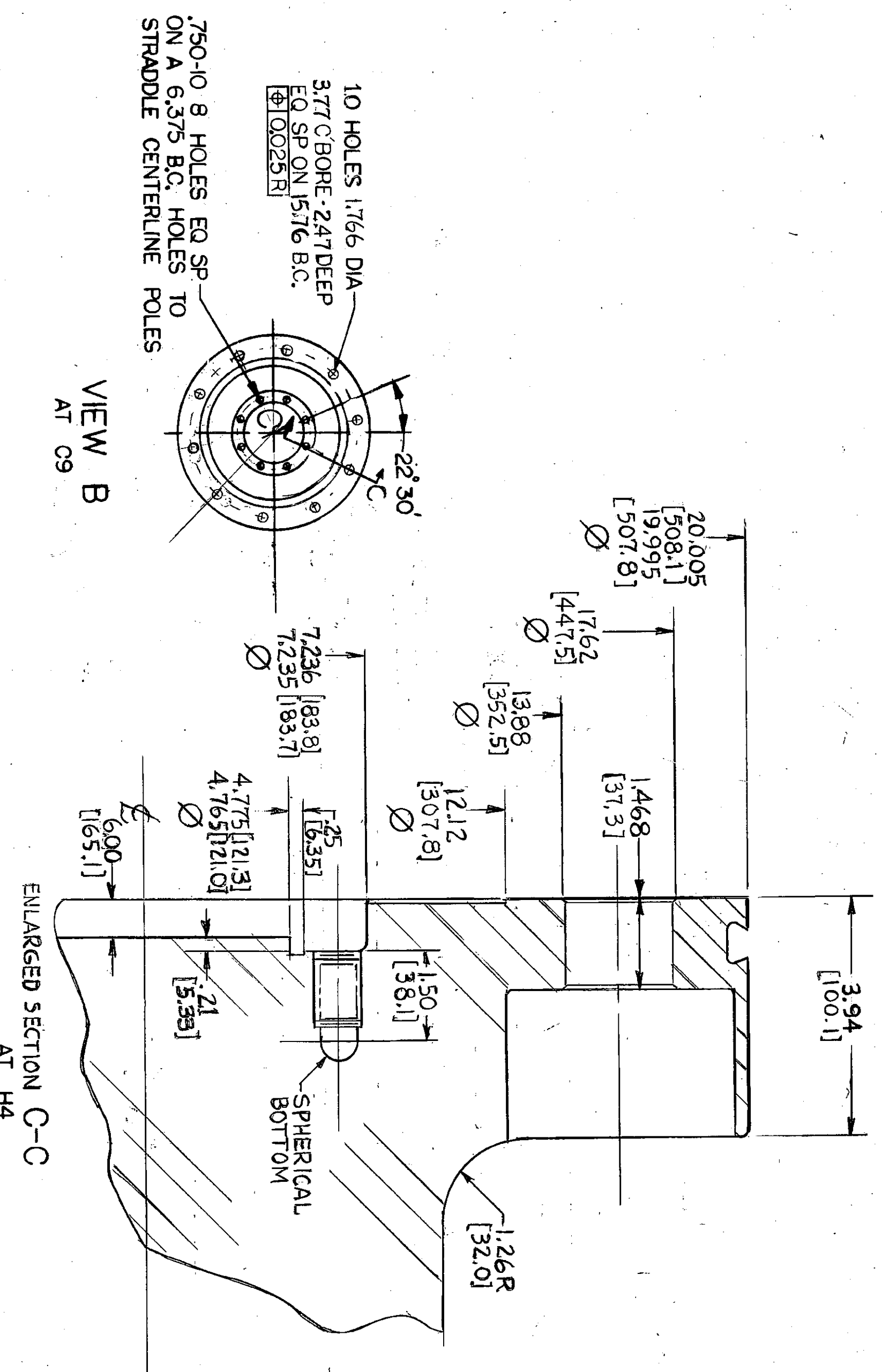
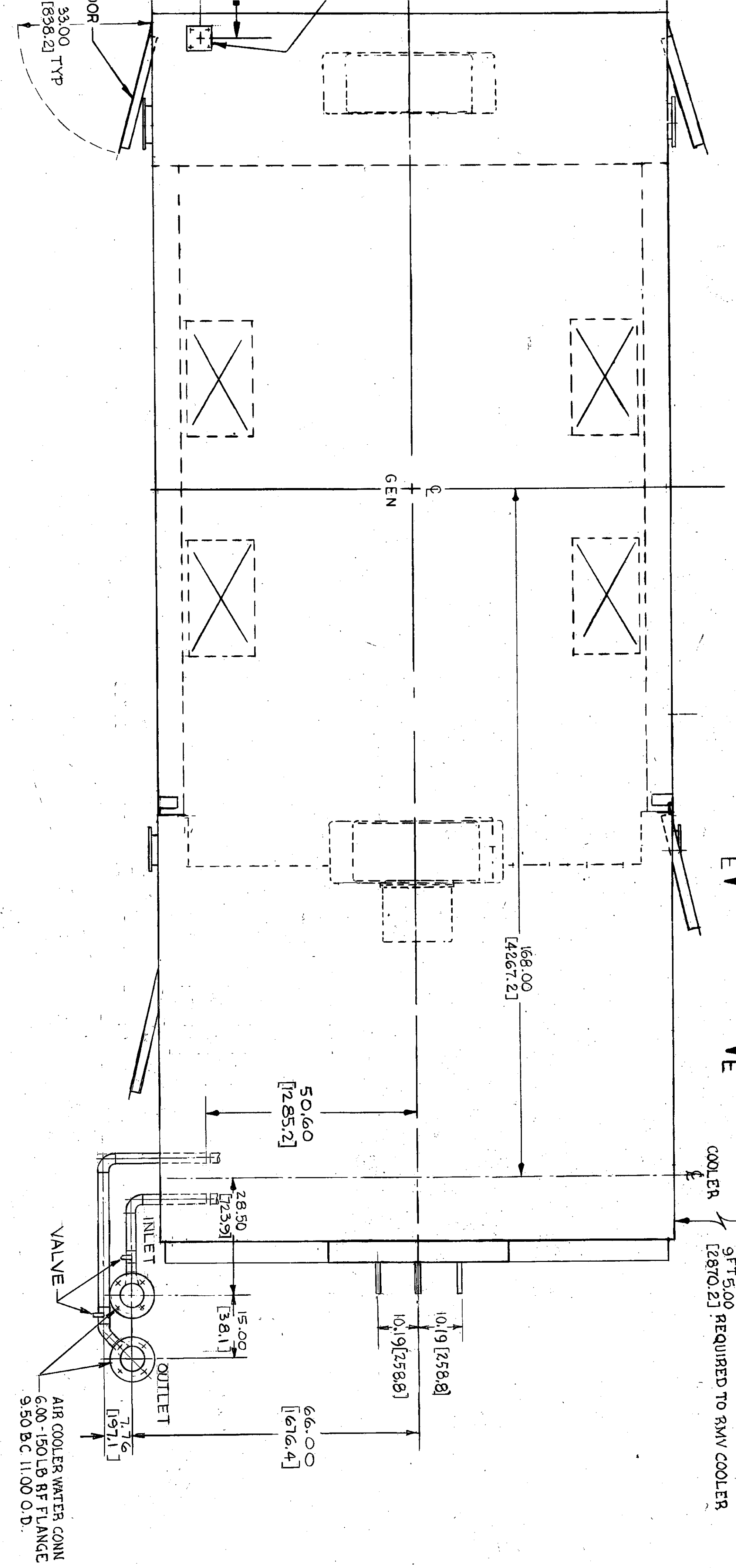


12 11 10 9 8 7 6 5 4

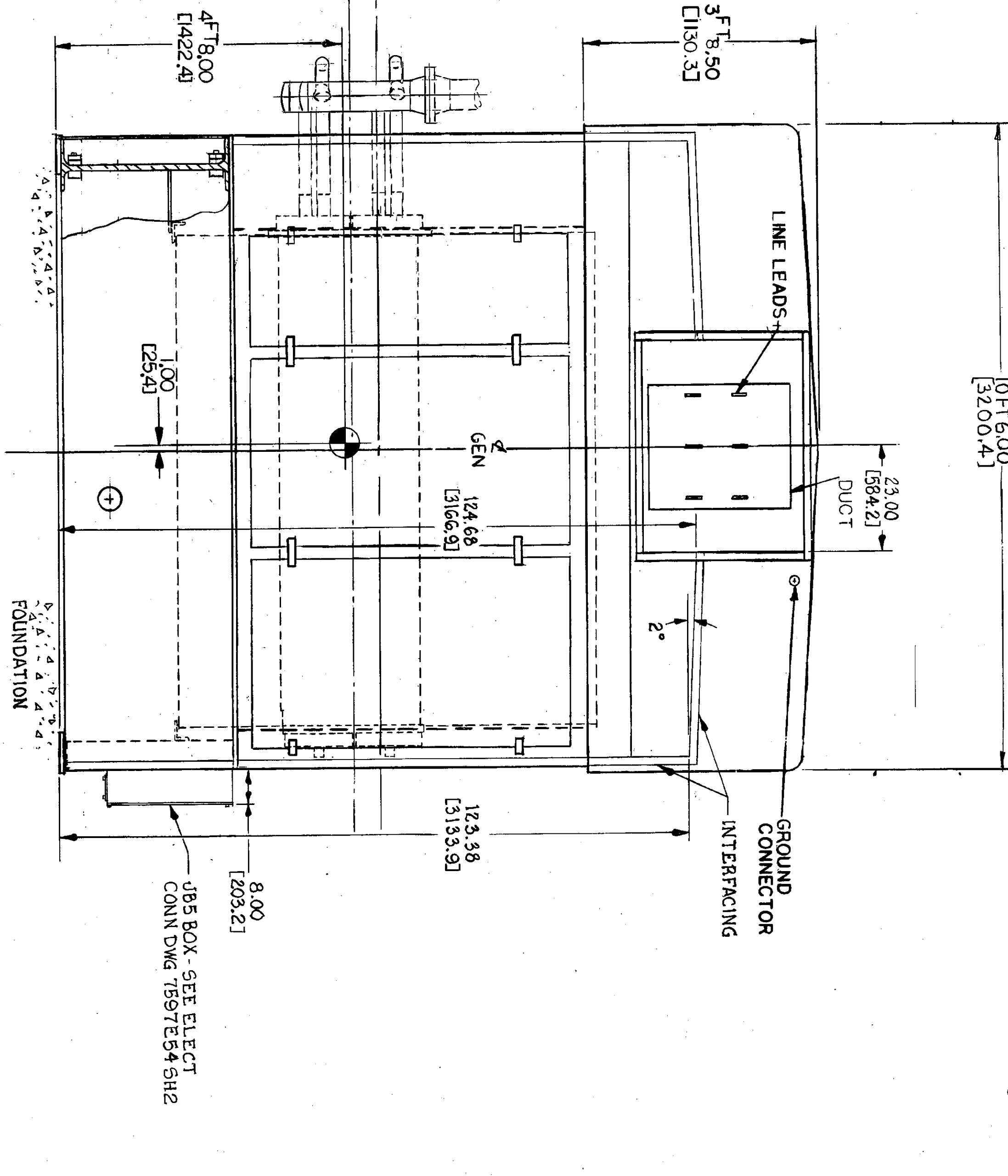
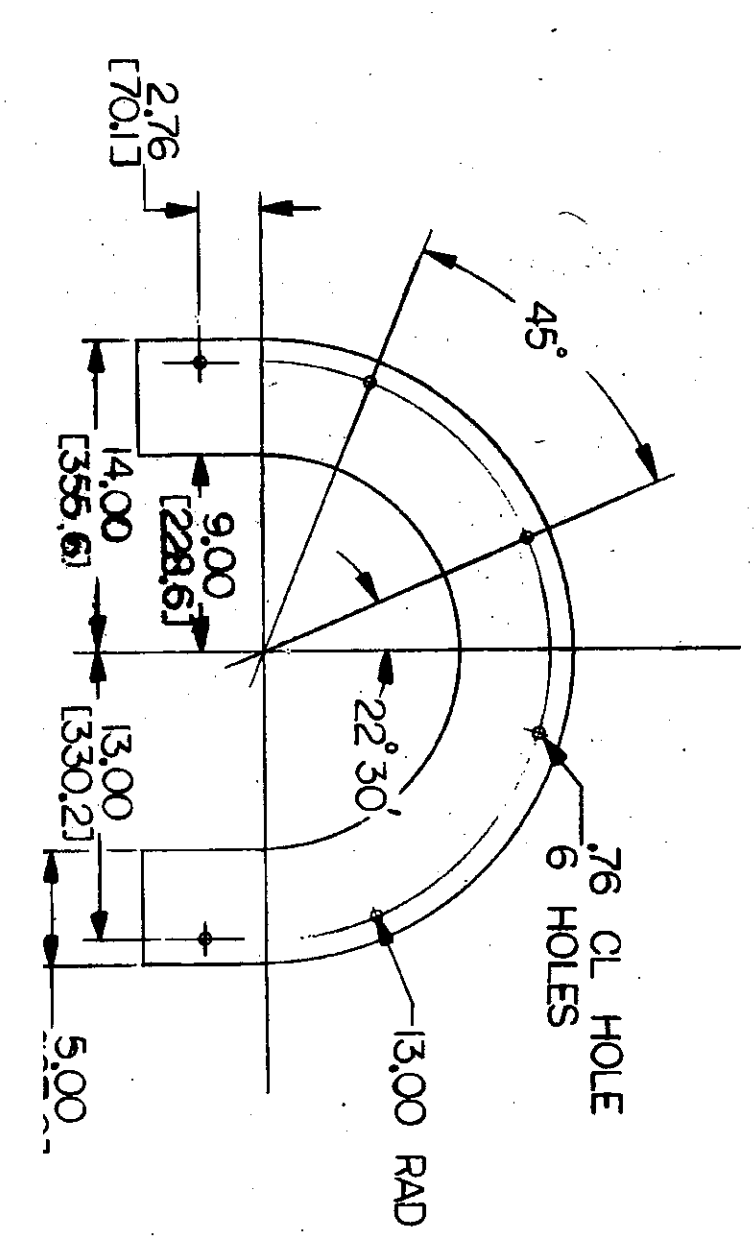
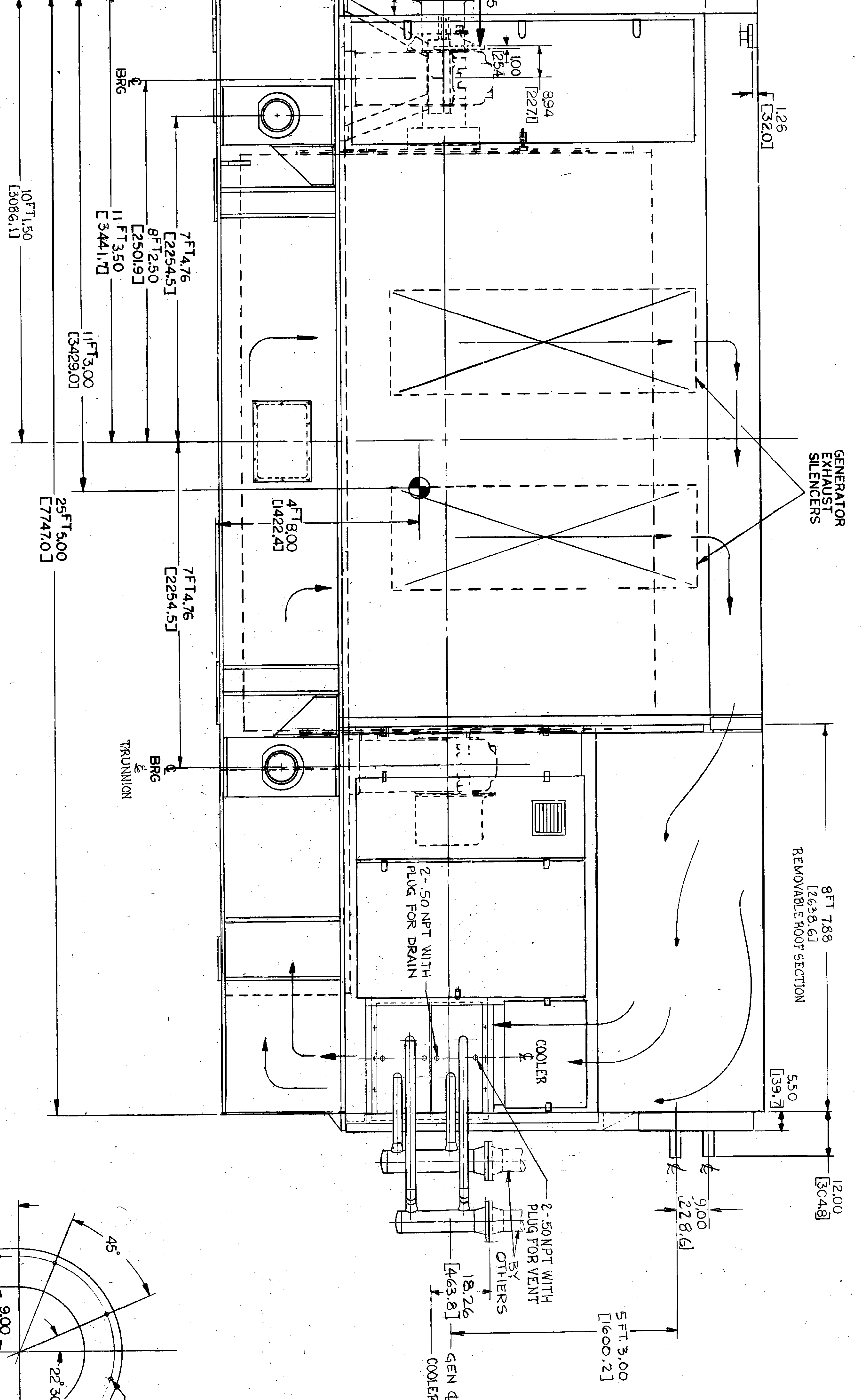








VIEW B  
AT 09



2. APPROXIMATE WEIGHTS

GEN. STATOR	GEN. ROTOR	MISC. REFRIG. S TWO	REFRIG. S TWO	COILERS	GEN. PACKAGE	TOTAL
94,500	24,500	4,000	28,000	3,000	173,000	
42,865.20	11,113.20	1,814.40	12,700.80	1,326.80	78,444.50	



## SECTION 2

### DESCRIPTION OF SURFACE CONDENSER

#### SIZE AND TYPE

The surface condenser supplied to this installation is a two pass, nondivided type. The shell section is rectangular in shape.

#### TUBE ARRANGEMENT

The tubes were expanded in the tube sheets at each end. Tubes at both ends are flush with the tube sheets. An arrangement of the tube ends with respect to the tube sheets can be found on the condenser General Arrangement Drawing E-32F-RG-501X1.

The diagrammatic tube sheet drawing (SC-23375) also will aid recording of leaky or plugged tubes.

#### SHELL EXPANSION ELEMENT

The shell is constructed with a diaphragm-type expansion element which will accommodate expansion or contraction movements of the tubes, and relieve unequal expansion strains between the shell and tube bundles.

#### INTERNAL SHELL ARRANGEMENT

The tube bundle is designed such that steam enters from periphery and center lane areas of the unit. It then passes through the tube array until it reaches a common area near the center of the bundle before entering the openings in the support plates (See CM-1129). The arrangement of tubes allows effective feeding of steam to all tubes. It creates a decreasing cross-sectional area of the steam lanes; such that, as steam penetrates the tube bundle, brisk velocities are maintained, maximum condensing efficiency is assured, and air blanketing is eliminated.

The tube support plates divide the condenser into a number of small compartments, which act as short length condensers placed end to end with each compartment having its own steam inlet. They are arranged so that the air and non-condensable vapors will be processed from the circulating water outlet end to the circulating water inlet end of the condenser. The air and non-condensable vapors from the outlet end compartment flow through the cutout in the support plate entering into the next compartment. From here this flow merges with the air and non-condensable vapors again exiting through the cutout in the next support plate. This same process occurs from compartment to compartment along the entire length of the condenser until the flow reaches the coldest end (circulating water inlet end.) In the circulating water inlet end compartment the processed mixture is

merged with the air and non-condensable vapors of this compartment and are finally led into the terminal cooler. In passing through the compartments the air and non-condensable vapors are cooled and a large part of the water vapor is condensed. This reduces the partial vapor pressure of the mixture. In this state it is drawn into the air removal equipment where it is eventually compressed to atmospheric pressure.

Each condenser is provided with air vapor outlet connections. These connections originate in the terminal cooler at the coldest end of the condenser. For all condensers it is recommended that permanent metal tags bearing proper designations be affixed to valve handwheels in the respective air vapor lines in order to avoid confusion when taking a half or quarter section of the condenser out of service.

### HOTWELL

The hotwell measures approximately 7'-4" wide X 21'-3" long and will maintain a storage of 780 gallons. Refer to General Arrangement Drawing for condensate level. There is a 20" diameter access openings on the end wall.

### WATERBOXES

The waterboxes are of the ring type construction on the inlet-outlet and bonnet on the return end. Circulating nozzles are 30" diameter. The waterboxes were designed for a pressure of 75 psig and shop tested at 112 psig. Waterboxes have vent and drain connections plus two 20" diameter quick-opening manholes per waterbox for inspection and cleaning.

The waterboxes have been coated internally with coal tar epoxy, Carbomastic #3 and #14. All protective coatings have certain application and operating limitations. They are subject to mechanical damage and we recommend periodic inspections. The coating should be completely covered and protected from damage during any tube work or cleaning. No welding or burning should be done on the waterboxes since this will damage the coating. The coating should be thoroughly inspected and tested for leaks or damage when all internal work is completed and just prior to placing the unit in service.

### CONNECTING PIECE

The connecting piece has two direct bolted flanges for direct turbine mounting. Access can be made into the connecting piece through a 20" diameter manway located on top. This area provides a smooth transition from the turbine exhaust to the condenser tube bundle.

DESIGN DATATubes

1.	Supplied by	Ecolaire
2.	Installed by	Ecolaire
3.	Main Section - O.D./BWG Quantity/Mat'l	3/4"/18BWG 3,668/ASTM B111/443 (Admiralty)
4.	Total Number of Tubes	3,668
5.	Length for Ordering	22'-5"
6.	Tolerance for Ordering	+3/16, -0

Performance

1.	Surface Area - (Sq. Ft.)	16,030
2.	Steam Load - (Lb./Hr.)	130,151
3.	Heat Load - (BTU/Hr.)	108,400,000
4.	Absolute Pressure - (In. Hg. Abs.)	1.5
5.	Cleanliness Factor - (%)	85
6.	Inlet Circ. Water Temp. - ( $^{\circ}$ F)	70.5
7.	Circulating Water Flow - (Gal./Min.)	13,366
8.	Water Velocity thru Tubes - (Ft./Sec.)	7.0
9.	Condenser Friction - (Ft. of Water)	17.45
10.	Effective Tube Length	22'-3"
11.	Tube Sheet Material	ASTM B171/365 (Muntz)
12.	Number of Support Plates	8
13.	Cooling Water Side Volume-(Gal.) Flooded Condition	3,254.24
14.	Steam/Condensate Side Volume-(Gal.) Flooded Condition	11,573.11



# EXPECTED CONDENSER PERFORMANCE

1. CIRC. WATER TEMPS. 65, 70 $\frac{1}{2}$ , 75, 81
2. CLEANLINESS FACTOR: 85%
3. 100 PERCENT UNIT HEAT LOAD: 108,400,000 #/HR
4. CIRC. WATER FLOW: 13366 GPM
5. EFFECTIVE SURFACE AREA: 16,030 FT<sup>2</sup>

